

AME0009

## Problems and Result in Torsional Vibration Measuring and Analyzing on Marine Diesel Engine Propulsion

LUU Do Duc <sup>[1]</sup>, SI Hoang Van <sup>[2]</sup>, VANG Le Van <sup>[2]</sup>

<sup>[1]</sup> Viet Nam Maritime University, [luudoduc@gmail.com](mailto:luudoduc@gmail.com) ;

<sup>[2]</sup> Ho Chi Minh City Transportation University, [si.hoangvan@gmail.com](mailto:si.hoangvan@gmail.com) ; [levanvang@gmail.com](mailto:levanvang@gmail.com)

### Abstract:

This paper presents problems and result of measuring and analyzing torsional vibration on marine diesel engine propulsion shaft. Authors mocked-up, simulate for measuring and processing multinomial torsional vibration signal with interference or noise; error cycle prediction when extract sample in area of realtime and frequency. The results of this paper has deployed in case of normal firing and misfiring in a marine diesel engine.

**Keywords:** Torsional vibration on marine diesel engine propulsion; Interference and sampling errors of torsional vibration measurement; Fast Fourier Transform (FFT).

### 1. Introduction

Torsional vibration problems arise simultaneously with intensive use of marine diesel engines for ship propulsion shaft. Calculating and measuring torsional vibration in marine diesel engine propulsion shaft must be stipulated in rules and regulation for the classification of ships by IACs (NK, ABS, Lloyd, Veritas, DNV, Russian Maritime Register,...).

We used the FFT Algorithm to process and analyze interference (noise) signals which come from speed encoder, air-fuel system, loading, operating conditions of engine or measuring device errors. Then show their important properties in area of realtime and frequency (Maximum, Minimum and Average values). In fact, there are many factors for measuring torsional vibration in marine diesel engine propulsion shaft line, even incase of diesel engine speed run rather stable, number of samples extracted for one working cycle of engine is alternator. Thus, error factor of extract sample is present and we need research to answer for questions:

- Does the measuring and processing torsional vibration signal error when extract sample error ?;
- How to overcome extract sample error when speed of diesel engine is variable ?.

That is the reason why this paper need be researched and found the results. During the research time, Authors mocked-up, simulated and programed on Matlab software tool when diesel engine fire normally and misfiring at one cylinder.

### 2. Error extract sample in measuring and processing torsional vibration signal

Extract sample error (sample rate) is consequence of set up configuration of collect sample error, the length of sample is not correct for one working cycle of diesel engine (include z cylinders). Ideal length of sample is covered seal for one working cycle of diesel engine, for 2 stroke engine, length is 360 degree of

crankshaft angle and for 4 stroke engine is 720 degree of crankshaft angle.

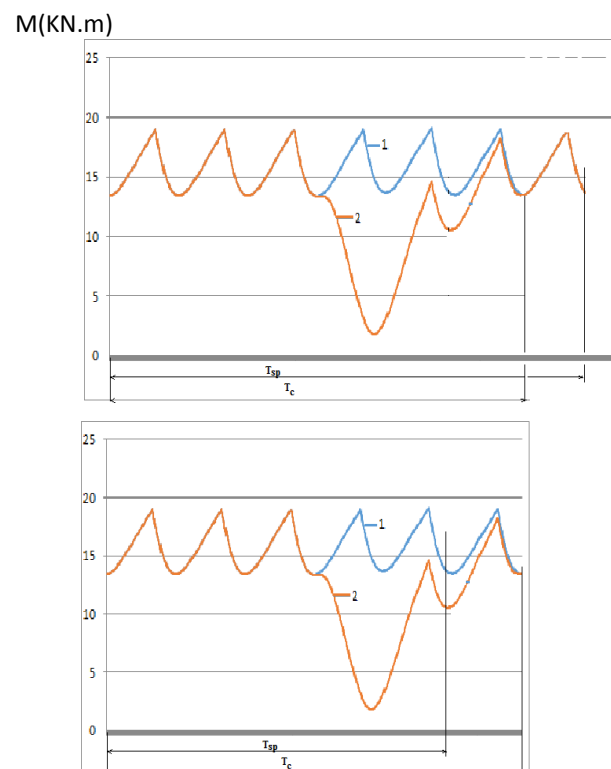


Fig 1. Extract sample for gas torque signal  
Line 1: Total torque when engine is fire normally;

Line 2: Total torque when engine is misfiring at one cylinder.

*Case (1):* Extract sample cycle  $T_{sp} > T_c$  or  $f_{sp} < f_c$ . In extract sample length  $N_{sp} = 1024$  samples,  $dt = 1/f_{sp} > dt_c = 1/f_c$  (s).

*Case (2):* Extract sample cycle  $T_{sp} < T_c$  or  $f_{sp} > f_c$ . In extract sample length  $N_{sp} = 1024$  samples,  $dt = 1/f_{sp} < dt_c = 1/f_c$  (s).

$f_{sp} = \text{const}$  and  $N_{sp} = 1024$  samples.

## AME0009

In 2 stroke engine, MAN B&W 6S46 MC-C, at speed 120 rpm, if extract sample error when diesel engine speed is variable within  $\pm 4$  revolution per minute (rpm), that means a range diesel engine speed is 116 - 124 rpm,  $N_{sp}$  is collect sample quantity and extract sample rate is constant  $f_{sp} = 2048$  sample/second.

$n_E$  (rpm) is real speed of diesel engine;

Time for extract 01 sample is  $dt_{sp} = 1/f_{sp} = 4,8828E-04$  (s) = 488,28 $\mu$ s;

Time for extract 2014 samples is  $T_{sp}$  (Sheet 1).

Example:  $n_E = 116$  rpm,  $\Delta\phi = 360 \cdot (60/n_E)/T_{sp}$ ,  $\Delta\phi = 372.4$  Crankshaft degree (1.03 Cycle)

**Table 1- Extract Sample Error when speed of diesel engine is variable**

N (Sample)	n (rpm)	fE	dtE(s/cki)	dt_sp	f_s (Sample/sec)	phi_sp	chki_sp
1024	116	1.93	0.517	5.051E-04	2048	372.4	1.03
	118	1.97	0.508	4.966E-04		366.1	1.02
	120	2.00	0.500	4.883E-04		360.0	1.00
	122	2.03	0.492	4.803E-04		354.1	0.98
	124	2.07	0.484	4.725E-04		348.4	0.97

### 3. The matters of noise filter and torsional vibration signal forecast

In digital signal processing books, there are many other digital filters. We apply some digital filters to filter interference or noise out measured torsional vibration signal and to predict difference of signal when extract sample error happens.

The simulation results shown in figure 2 and figure 3.

(1)- Filtering interference out torsional vibration signal measured (Fig 2);

(2)- Predicting difference of torsional vibration signal when extract sample error happens (Fig 3).

After researching application of some filter in Matlab tool such as Traditional Filter and Adaptive filter. Output signal moves to one direction in the traditional filter so they is not suitable for our matter. On the contrary, the adaptive filter predicts signal for working cycle of diesel engine, gather data when lack or cut down data when redundant and re-calculate new value by linear interpolate method.

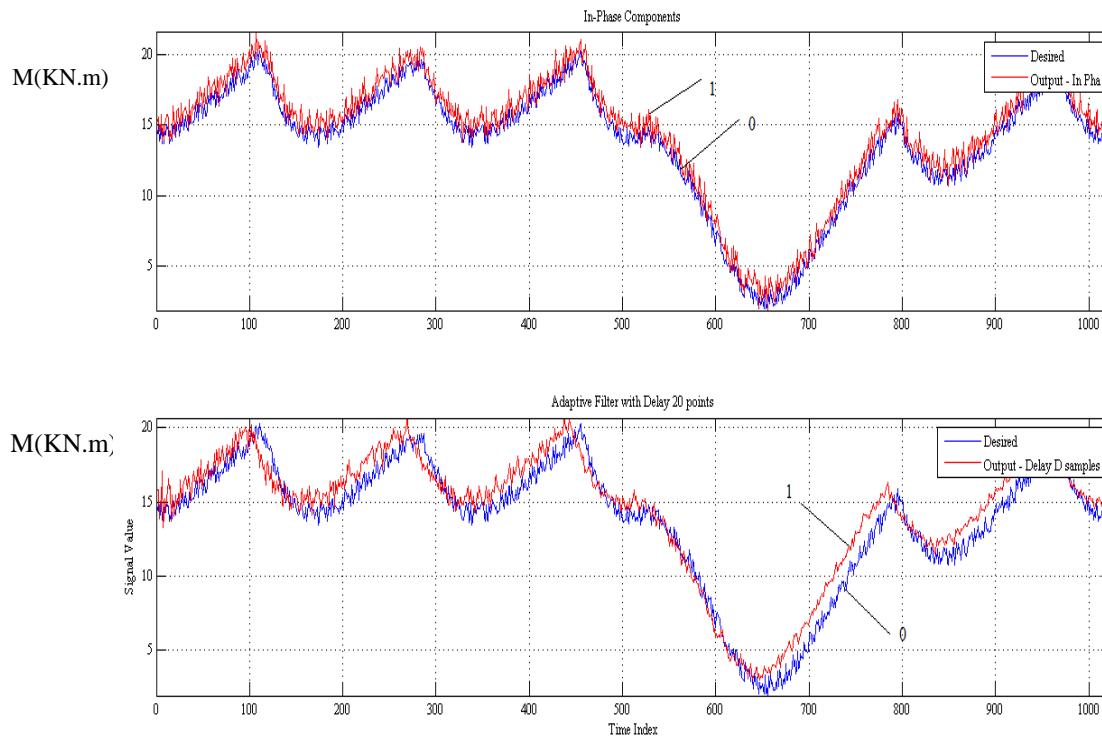


Fig 2. Torsional vibration signal simulation use the adaptive filter

Fig 2: Line 0 (blue) - Torque with interference;  
Line 1 (red) - Torque with interference filtered

## AME0009

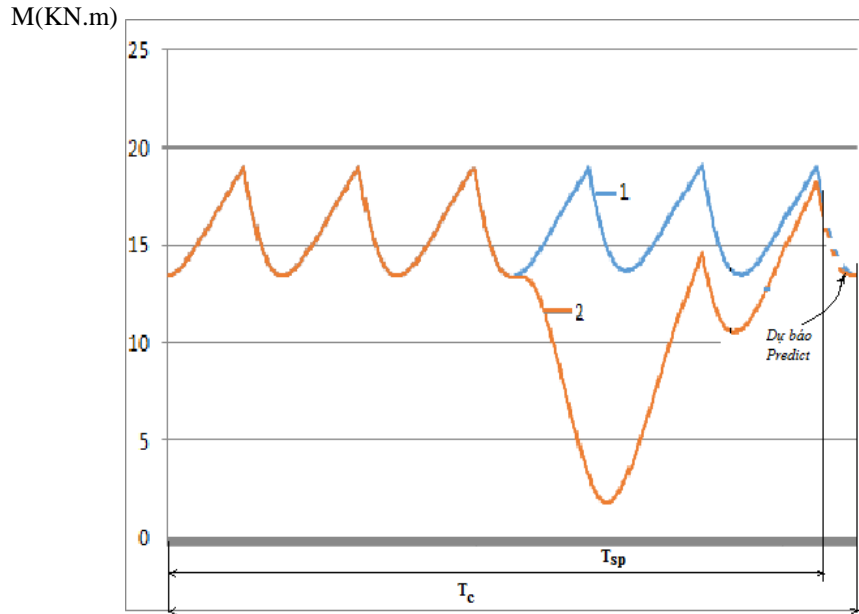


Fig 3. Torsional vibration signal prediction when extract sample cycle is shorter than diesel engine cycle

Fig 3: Line 1 (blue) - Torque when diesel engine work normally;  
Line 2 (yellow) - Torque when engine is misfiring at one cylinder.  
Continuous line: signal measured; Dash line: predict signal after measuring signal at continuous line.

### 4. Analysis of Results

In the table 2, when diesel engine works normally, in principle total torque contains main harmonics and secondary harmonic is zero. However, due to calculate error, to round in calculate process, random interference will exist so the amplitude vector will occur and secondary harmonic value is different zero. Extract sample error is lead to amended error for low grade harmonics that is hard to recognize easily. So we only consider to main harmonic (grade 6-H6) which is alternate wide and is easy to realize and estimate results shown in table 3.

Table 2. Simulation result for total torque $M_0$ with 12 harmonics of amplitude vector $R_0$ and phase 0, without interference, extract sample error, diesel NORMAL												
$R_0$	0.0775	0.0656	0.0531	0.0479	0.0504	2.3440	0.0170	0.0108	0.0116	0.0119	0.0185	0.5966
n=116	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_c$	0.1284	0.1878	0.1067	0.2377	0.4010	2.1746	0.5800	0.2768	0.1790	0.1695	0.2012	0.4779
$R_0 - R_c$	-0.0509	-0.1221	-0.0536	-0.1898	-0.3507	0.1694	-0.5630	-0.2660	-0.1674	-0.1576	-0.1827	0.1187
n=118	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_c$	0.0962	0.1395	0.0470	0.1504	0.2505	2.3094	0.2579	0.1341	0.0801	0.0867	0.1243	0.5651
$R_0 - R_c$	-0.0187	-0.0739	0.0061	-0.1025	-0.2001	0.0345	-0.2409	-0.1232	-0.0685	-0.0748	-0.1058	0.0314
n=120	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_c$	0.0775	0.0656	0.0531	0.0479	0.0504	2.3440	0.0170	0.0108	0.0116	0.0119	0.0185	0.5966
$R_0 - R_c$	-0.0000	0.0000	-0.0000	0.0000	0.0000	0.0000	-0.0000	0.0000	-0.0000	-0.0000	0.0000	0.0000
n=122	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_c$	0.1203	0.0322	0.1528	0.1363	0.2490	2.2941	0.1993	0.0922	0.0871	0.0845	0.1395	0.5544
$R_0 - R_c$	-0.0428	0.0334	-0.0997	-0.0884	-0.1987	0.0498	-0.1823	-0.0814	-0.0755	-0.0726	-0.1210	0.0422
n=124	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_c$	0.1592	0.0812	0.2253	0.2474	0.5217	2.1949	0.3561	0.1818	0.1488	0.1503	0.2866	0.4721
$R_0 - R_c$	-0.0817	-0.0156	-0.1723	-0.1996	-0.4714	0.1490	-0.3391	-0.1710	-0.1372	-0.1384	-0.2681	0.1245

## AME0009

n(rpm)	116	118	120	122	124
dR(6)	0.219	0.062	0	0.022	0.090
dR(6)%	9.3%	2.6%	0.0%	0.9%	3.8%

Similar to consider total torque signal when engine work normal, random interference Arand = 1.5 (around 9-10% compare with maximum torque). We consider for 2 main harmonics dR(6) and dR(12) with results as per table 4 and table 5 below:

R <sub>0</sub>	0.0775	0.0656	0.0531	0.0479	0.0504	2.3440	0.0170	0.0108	0.0116	0.0119	0.0185	0.5966
n=116	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
R <sub>c</sub>	0.1342	0.2065	0.1140	0.2341	0.4251	2.1911	0.5842	0.2582	0.1861	0.1389	0.2438	0.4763
R <sub>0</sub> -R <sub>c</sub>	-0.0567	-0.1409	-0.0610	-0.1862	-0.3748	0.1528	-0.5672	-0.2473	-0.1746	-0.1269	-0.2253	0.1202
n=118	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
R <sub>c</sub>	0.1059	0.1588	0.0548	0.1493	0.2739	2.3331	0.2610	0.1093	0.0814	0.0631	0.1549	0.5543
R <sub>0</sub> -R <sub>c</sub>	-0.0284	-0.0931	-0.0018	-0.1014	-0.2235	0.0108	-0.2440	-0.0985	-0.0699	-0.0512	-0.1364	0.0423
n=120	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
R <sub>c</sub>	0.0775	0.0656	0.0531	0.0479	0.0504	2.3440	0.0170	0.0108	0.0116	0.0119	0.0185	0.5966
R <sub>0</sub> -R <sub>c</sub>	-0.0000	0.0000	-0.0000	0.0000	0.0000	0.0000	-0.0000	0.0000	-0.0000	-0.0000	0.0000	0.0000
n=122	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
R <sub>c</sub>	0.0913	0.0846	0.0537	0.0543	0.0734	2.3724	0.0127	0.0155	0.0255	0.0414	0.0587	0.5819
R <sub>0</sub> -R <sub>c</sub>	-0.0138	-0.0189	-0.0007	-0.0065	-0.0230	-0.0284	0.0043	-0.0047	-0.0139	-0.0295	-0.0402	0.0147
n=124	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
R <sub>c</sub>	0.1303	0.0239	0.1521	0.1393	0.2402	2.324	0.1989	0.1142	0.1006	0.0916	0.1755	0.5426
R <sub>0</sub> -R <sub>c</sub>	-0.0528	0.0417	-0.0991	-0.0915	-0.1898	0.0196	-0.1819	-0.1034	-0.0891	-0.0796	-0.1570	0.0540

n(rpm)	116	118	120	122	124
dR(6)	0.1528	0.0108	0	-0.028	0.020
dR(6)%	4.6%	0.3%	0.0%	-0.8%	0.6%
dR(12)	0.1262	0.0423	0	0.0147	0.0540
dR(12)%	21.2%	7.1%	0.0%	2.5%	9.1%

When signal with interference, error level dR(6) reduces than case of without interference. However, speed level is variable within  $\pm 4$  rpm, at node 1 (dR6), harmonic error is near 5%. On the contrary at node 2 (dR12) is 21.2%.

Finally, we researched on diesel engine in case of misfiring at one cylinder and the results shown the same in table 6.

## AME0009

$R_{MF1}$	4.3275	2.7333	1.7725	1.3345	0.9192	1.8486	0.4560	0.3194	0.2347	0.1483	0.1180	0.5212
n=116	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_{MF1} - R_{MFC}$	0.1494	0.0142	-0.1112	0.1937	-0.4066	0.2516	-0.0618	-0.1333	-0.1056	-0.0171	-0.1683	0.1468
dR%	3.45	0.52	-6.28	14.52	-44.24	13.61	-13.56	-41.72	-44.99	-11.53	-142.63	28.16
n=118	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_{MF1} - R_{MFC}$	0.0778	0.0020	-0.0701	0.1166	-0.2366	0.0976	0.0475	-0.0430	-0.0459	0.0245	-0.0742	0.0272
dR%	1.80	0.07	-3.95	8.74	-25.74	5.28	10.42	-13.45	-19.56	16.53	-62.90	5.22
n=120	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_{MFC}$	-0.0053	-0.0195	-0.0074	0.0018	0.0004	0.0357	0.0034	-0.0079	-0.0066	0.0156	0.0341	-0.0340
$R_{MF1} - R_{MFC}$	-0.12	-0.71	-0.42	0.14	0.044	1.94	0.74	-2.464	-2.804	10.544	28.94	-6.55
n=122	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_{MF1} - R_{MFC}$	-0.0914	-0.0492	0.0649	-0.1321	0.2774	0.0535	-0.0926	-0.0138	0.0093	-0.0436	0.0474	-0.0116
dR%	-2.11	-1.80	3.66	-9.90	30.18	2.89	-20.30	-4.31	3.95	-29.40	40.17	-2.22
n=124	H1	H2	H3	H4	H5	H6	H7	H8	H9	H10	H11	H12
$R_{MF1} - R_{MFC}$	-0.1649	-0.0690	0.1151	-0.2448	0.5434	0.1154	-0.1713	-0.0612	0.0353	-0.1137	-0.0734	0.0702
dR%	-3.81	-2.52	6.49	-18.35	59.12	6.24	-37.57	-19.16	15.05	-76.64	-62.21	13.46

Base on data of table 6, we realized that the harmonic amplitude errors change with a large range even the diesel engine speed is not change ( $n = 120$  rpm); the harmonic amplitude (H10) is 10.544% and the harmonic amplitude (H11) is 28.9%. The harmonic amplitude error at secondary harmonic is higher than at main harmonic (H6, H12) also when there is one misfiring cylinder on diesel engine.

### 5. Conclusion

This paper has outlined the use of Matlab software to simulate measuring and processing of torsional vibration signal and estimate the effect of interference and extract sample error incase of engine work normal and misfire.

Digital filters can be used to reduce the interference in the frequency modulator carry signal and predict geometric length of extract sample cycle.

### REFERENCES

1. E.O Brigham, (1988) *The fast Fourier transform and its applications*; Prentice Hall Signal Processing Series, Englewood Cliffs, NJ
2. Đỗ Đức Lưu (2009). *Dynamic and Diagnostic marine diesel engine by vibration*. NXB. GTVT. Hà Nội.
3. Đỗ Đức Lưu (2005). *Một số vấn đề trong nghiên cứu, tính dao động xoắn hệ trục chính diesel lai chân vịt trên tàu thủy hiện đại*// Tạp chí GTVT. 2005. №10. Tr.32,33,62,63.
4. D.D.Luu, H.V.Si, L.V.Vang. (2014). *Quy chuẩn Việt Nam về dao động xoắn hệ trục diesel và ứng dụng xây dựng phần mềm tự động tính giới hạn xoắn các thành phần hệ trục diesel lai máy công tác*. Technology Science – Maritime Magazin, No 40, Maritime Publisher.
5. D.D.Luu, H.V.Si, L.V.Vang. (2014). *Virtual Instrument to automatically draw the permit maximum torsional pressures for the marine diesel shaft system*. Technology Science – Maritime Magazin, No 40, Maritime Publisher.
6. Nguyen Quoc Trung. (2006). *Digital Signal Processing and filter*. Ha Noi Science and Technic Publishing House.
7. Matlab Tool USA, R2014a